# Cost – Effective Solution for Twist Free Chassis Design of Three Wheeler Vehicles

### SatishChoudhar

Department of mechanical engineering, VidyaPratisthan's college of engg.
Baramati, Maharashtra, 413133, India

### DadasahebRupanwar

Department of mechanical engineering, VidyaPratisthan's college of engg.
Baramati, Maharashtra, 413133, India

### Ashok kalgunde

Department Operations, Piaggio Vehicles private limited, Baramati, Maharashtra, 413133, India

Abstract - This paper presents studies and analysis for designing the solution against chassis twist in three wheelers. It covers all comprehensive aspects of chassis twist, measurement, causes and effects of twist, stresses; energy consumption manpower cost etc. The project also covers the designing of standard operating procedure for quality production, statistical quality control approach, and also designing the manufacturing process for higher productivity and quality.

The mechanism of twist in chassis and analysis in terms of twist repair, manpower cost, electrical energy consumption, stresses is analyzed. The manpower cost reduction is by 48.34 %, whereas the electrical energy cost is reduced by 42.69 %.

By designing the SOP. (Standard operating procedure) at work stations leads to increase in awareness for quality and decrease of rework rate by 75 % at work stage, where main beam subassembly is produced.

The statistical quality control approach can be implemented for the quality aspects and its control. The cost saving up to 66.49 % can be achieved by implementing new method for the chassis going to CED paint shop as part for productivity improvement. Also the cost saving is by Rs. 12.80 per chassis by implementing new method.

Keywards: chassis, chassis twist, stress analysis, standard operating procedure, statistical quality control, Productivity, quality, cost, brazing

### I. INTRODUCTION

The motor vehicles, both passenger car and goods truck, are generally considered to be made up of two major assemblies 1) chassis 2) Body. Chassis is a French term and was initially used to denote the frame or the main structure of vehicle. It is now extensively used to denote the completely vehicle except the body for the heavy vehicle having a separate body. In case of light vehicles of monoconstruction, it denotes the whole body except the additional fittings in the body.

### A. Objectives of study

- 1. To design for the solution of chassis twist for three wheelers.
- 2. To increase the quality rate for the production of three wheelers chassis as a product.
- 3. To increase the customer satisfaction level by decreasing defects in chassis of three wheelers as a product.
- 4. To design the manufacturing process for higher productivity and quality in three wheelers without twist.

#### B. Twist in chassis

- Twist in chassis is specified as a) dome twist b) Rear or overall twist.
- The height difference between left and right side measured at front dome area of chassis is called as dome twist.

- The height difference between left and right long member measured at rear side is called as rear or overall twist.
- The causes and effects of twist in chassis are given and comprehensive aspects of twist mechanism to be analyzed

### C. The causes for twist in chassis

- Clamping pressure low- The certain assemblies are produced on the fixture with pneumatic clamping system.
- 2. The blocks of the fixture wear out or the loose. Dome frontal pin location not ok.
- 3. The intermediate box wrongly fitted to the long member.
- 4. BS2 / BS3 model changeover not done at work stage where long member and cross member assembly are joined.
- 5. Main frame assembly angle variation or unsymmetrical welding of front parts.
- 6. The work stages operators are not of qualified skill or lack of technical training for the work.
- 7. The dimensional variation in the incoming materials. Scarcity of measuring gauges and working fixtures preventive maintenance adherence.

#### Effects of twist in chassis

- 1) Aesthetic design considerations affected.
- 2) The wear of tyre surfaces is excess.
- 3) It needs excess physical efforts and energy, time and the resources for rectification of twist in chassis.
- 4) The vehicle may get skid due to uneven forces.
- 5) The rubber shock springs get unevenly deflected.
- 6) The rapid failure of chassis and customer dissatisfaction increases.
- 7) The dimensional variation occurs tending to mismatches of parts during production of chassis, the dimensional accuracy not achieved. The cost associated with repair of chassis is increased.

Table 1 Dome twist data for period

Sr.No.	Month	Average twist	Power cost (Rs)
1)	January 2014	11.7	9.0
2)	Feb. 2014	9.4	7.23
3)	March 2014	7.7	5.93
4)	April 2014	6	4.62
5)	May 2014	5	3.85

### II. PRPPOSED METHODOLOGY

#### A . Controlling Of Twist InChassis

### A1.ParallismControlled By Pneumatic Cylinder Mechanism

The pneumatic cylinder mechanism is provided for controlling parallism of front parts. It is working on pneumatic operating pressure of 6 bar . The clamping force = 1061.1 N.

The calculation is given as below.

Pneumatic piston force,  $F = A^* P - R$  (1)

 $F = P*10*d^2*3.14 / 4 - R$  (2)

Where A = Piston area (cm²), P= Operating pressure (bar), R = Friction force = 10 % (N) Piston area A =  $3.14*d^2/4$ , d = piston diameter = 50 mm = 5 cm.

Hence  $A = 19.625 \text{ cm}^2$ 

F = 6\*10\*5\*5\*3.14/4 - 106.1

= 1177.5 -117.8

= 1059.7 N



Pneumatic cylinder

Figgure 1.Parallism controlled by pneumatic cylinder mechanism

### III. EXPERIMENTATION AND RESULT

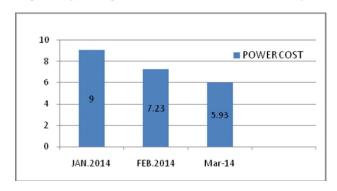
### A. Kaizen Implementation For Holding Cup On bracket:



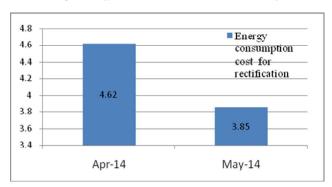
Clamp Miler for holding cup knob

Figure 2. Kaizen implementation for holding cup on bracket

Graph 1 Energy consumption cost for rectification (a), (before controlling)4/13



Graph 2 Energy cost for rectification (b), (after controlling)



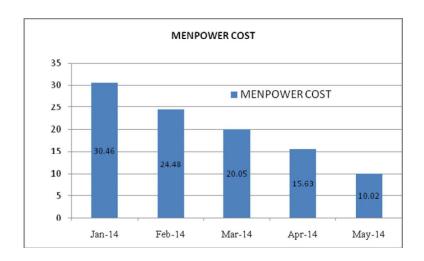
### B.Manpower Cost Analysis

The observations show that the operator time required for twist rectification is about 5 min. The salary paid to operator is of Rs 7500 per month. The following tabulated data is given here for the cost.

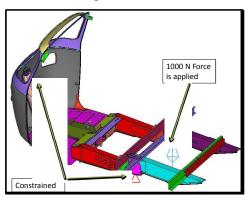
Table 2 manpower cost analysis

Sr no.	Month	Avg. twist	Total time(Min.)	Manpower cost(Rs.)
1	Jan.2014	11.7	58.5	30.46
2	Feb.2014	9.4	47	24.48
3	Mar.2014	7.7	38.5	20.05
4	April 2014	6.0	30	15.63
5	May 2014	3.85	19.25	10.02

Graph 3 Manpower cost for the given period



Twisting of chassis.



### C. CAE Analysis of Twisting Of Chassis:

- For analysing chassis in twisting condition in three wheeler, front end and one of the rear end is fixed
- Force of 1000 N is applied at the rear free end of the 3 wheeler chassis, tempting body to get twisted.
- After applying above condition, results are obtained for the displacement and stressof twisted body of the threeWheeler.

### SOFTWARES USED:

- 1) Modelling: Pro-E 2) FE Modelling: Hyper mesh 12.0
- 3) Solver and Post Processor: Optistruct and Hyper view

### ST 45 MATERIAL PROPERTIES:

- 1) Young's Modulus: 2.1E5 MPA 6/13
- 2) Poisson's ratio: 0.3,3) Density (Steel): 7.89e 6 Kg/mm<sup>3</sup>

### **MESHING PROPERTIES:**

1) Element Size –  $10 \ mm$  , 2) Element type – trias and quads (2D  $\,$  Rigid, bars and beams (1D)

Twisting Of Chassis (Stress Analysis)

As shown in below fig. front end and one of rear end is kept fixed and 1000N force is applied on the rear free end of the chassis

;)

### Twisting of chassis.

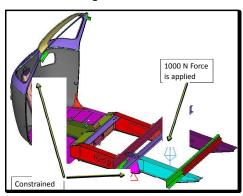
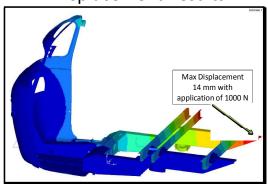


Figure 3.CAE Analysis (Twisting of chassis)

<u>Displacement Result</u>: Displacement of the rear end of the chassis at which force is applied. It is found around 14mm.

<u>Stress Result</u>: with application of 1000N force at one of rear end the stresses are found on rear end of vehicle. Max stress found is around 220 MPA which is less than yield limit (230 MPA)

## **Displacement Results**



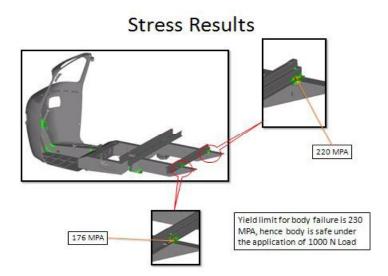
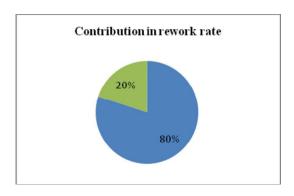


Figure. 5 CAE Analysis (stress result )

Table 3 Impact Of Implementation Of S.O.P.(standard operating procedure)

SR.NO	PARAMET ER	BEFORE IMPLEME - NTATION OF S.O.P.	AFTER IMPLEMENT- ATION OF S.O.P.
1)	Rework rate ( Qty / week )	High. Quantity = 4	Low and Quantity =1. The % decrease in rework rate=75%. Wastage control is achieved.
2)	Quality awareness	Less	More



Graph 4 Pie chart for analysis of rework rate

Pie chart is used to display the contribution of rework rate before and after implementation of standard operating procedure .From the tabulated data, 1 job /week contributes to 20 % (less rework rate), whereas, 4 jobs / week contributes to 80 % (before implementation of S.O.P.)

Table 4 Production, Defects And % Quality Product Rate

Sr.N	shift	production	No. of	% defective	% Quality product rate
о.	Sinit	production	defective	units	70 Quanty product rate
			defective		
1	1	167/175	40.	23.95	76.04
	2	73/73	12	16.44	83.56
2	1	175/175	16	9.14	90.86
	2	70/73	5	7.14	92.86
3	1	175/175	14	8.0	92.0
	2	65/73	12	18.46	81.54
4	1	175/175	18	10.28	89.72
	2	73/73	4	5.48	94.52
5	1	175/175	23	13.14	86.86
	2	73/73	6	8.22	91.78
6	1	175/175	9	5.14	94.86
	2	70/70	5	7.14	92.86

### FORMULAE FOR P- CHART

1) UCL = 
$$p+3$$
  $p.q$   $N$ 

2) LCL = 
$$\overline{p}$$
-3  $\overline{p}$ .q N

- 3)  $\bar{p} + q = 1$
- 4)  $\overline{p} = \sum_{n} p$

Notations have usual meaning as below:

- 1)UCL= upper control limit
- 2) LCL =lower control limit
- 3) N = average lot size
- 4) n = no. of lots.

Table 5 Lot wise Defect And U.CL., L.C.L. Data

	A	В	C	D	E
LOT1	240	52	0.22	0.1823	0.05763
LOT2	245	21	0.09	0.1823	0.05763
LOT3	240	26	0.11	0.1823	0.05763
LOT4	248	22	0.09	0.1823	0.05763
LOT5	248	29	0.12	0.1823	0.05763
LOT6	245	14	0.06	0.1823	0.05763
Total	1466	164	0.67	0.1823	0.05763

In table 5, following notations are used.

A = PRODUCTION,

B = DEFECTS ,

C = % DEFECTS,

D = U.C.L. (UPPER CONTROL

LIMIT)

E = L.C.L. (LOWER CONTROL

LIMIT |)

Graph 5  $\,$  P- Chart For Variable Lot Size:

Table 6 Brazing Alloy Consumption And Cost Analysis

SR.	PARAMETER	PREVIOUS	NEW
NO		METHOD (A)	METHOD (B)
1	Material used	Brazing filler alloy.	Brazing filler alloy with thumb grade sealer.
2	Cost	RS. 770/kg	sealant cost = Rs 160/kg
3	Consumption		
	analysis	4 Rods = 5 Chassis	1 Rod = 5 Chassis
		5 kg = 158 qty	20  Rods = 0.63  kg = 100  chassis.
		80 Rods = 100 Chassis	RS for 0.63 kg= Rs 485
		Also 80 Rods = 2.53 kg	20 lines sealant = 1 kg
			1 kg sealant = 100 Chassis
		RS = 1925	$Total\ Cost = 485 + 160$
			= RS 645
4	Cost for 100		
	Chassis	Rs.1925	Rs 645
5	Saving		Rs 1280



D. Designing The Manufacturing Process For Higher Productivity And Quality Without Twist

2500
2000
1500
1925
COST IN RS. FOR 100 CHASSIS

1000
645
Method A
Method B

Graph 6 Comparison of Consumables Cost



Brazing operation Figurer 6 (Previous Method) Section Of Chassis Where Brazing

### Is Applied



Figure 7 (New And Implemented Method) Section Of Chassis Where Along With Brazing, Thumb Grade Sealer Is applied.

Ratio Analysis: It is important technique to compute percentage saving on Material.

The percentage saving on material cost =

(Total cost by using brazing filler alloy) -

(Total cost by using brazing filler along with thumb grade sealer) \*10

(Total cost by using brazing filler alloy)

= 1280/1925 \*100

= 66.49 %

#### IV. CONCLUSION

- 1) The mechanism of twist in chassis and analysis in terms of twist repair, manpower cost, electrical energy consumption, stresses is analyzed. The electrical energy cost reduction is by 42.69 %, where as the manpower cost is reduced by 48.34 %.
- 2) By designing the SOP. (Standard operating procedure) at work stations leads to increase in awareness for quality and decrease of rework rate by 75 % at work stage, where main beam subassembly is produced.
- 3) The statistical quality control approach can be implemented for the quality aspects and its control.
- 4) The cost saving up to 66.49 % can be achieved by implementing new method for the chassis going to CED paint shop as part for productivity improvement. Also the cost saving is by Rs. 12.80 per chassis by implementing new method.

#### REFERENCES

- Yongjie Lu, Shaopu Yang, Shaohua Li, Liqun Chen, "Numerical and experimental investigation on stochastic dynamic load of a heavy duty vehicle" Elsevier Journal 2009.
- [2] JunboJia, Anders Ulfvarson "Dynamic analysis of vehicle deck interactions" Elsevier Journal 2005.
- [3] JunboJia, Anders Ulfvarson "Structural behavior of high tensile deck using trapezoidal stiffeners and dynamics of vehicle deck interactions" Elsevier Journal 2005.
- [4] A. Purushotham "Static stress and deflection analysis of a three wheeler chassis", Journal of Engineering Science and Technology, 2013
- [5] Vijaykumar V. Patel, R. Patel "Structural analysis of a ladder chassis frame" World Journal of science and Technology, 2012.
- [6] Steven Tebby, EbrahimEsmailzadeh and Ahmad Barari "Methods to determine torsional stiffness in an automotive chassis" Computer aided design and applications, PACE, 2011
- [7] Dr. Kripal Singh, Automobile engineering, Standard publishers, New Delhi.
- [8] R.B. Gupta, Automobile engineering, Satyaprakashan, New Delhi.
- [9] Nash Operational manuals for spot welding machines
- [10] Nash manual for CO2 welding machine, Nash robotics and automation pvt. Ltd.
- [11] Festo AG and CO, product range manual for cylinder and cylinder accessories.
- [12] R. Panneerselvam, Research Methodology, PHI Learning Pvt. Limited, New Delhi, 2010.
- [13] AmitabhaGhosh, Ashok Kumar Mallik, Manufacturing Science, Affiliated East-West Press Pvt. Ltd, New Delhi, 1995.
- [14] P.C. Sharma, production Engineering, S.Chand and company Ltd, New Delhi, 1997.
- [15] R.K.Jain, Engineering metrology, Khanna Publishers, Delhi, 1995.